#### ANNOUNCEMENT

# ANNOUNCEMENT

## OPEN BOUNDARY CONDITION (OBC) MINISYMPOSIUM

In conjunction with the seventh International Conference on Numerical Methods in Laminar and Turbulent Flow that will be held at Stanford University (Palo Alto, California) in the summer of 1991, we are pleased to announce a one-day pre-conference minisymposium on OBCs for viscous incompressible (laminar) flow. The purpose of the meeting is to compare numerical methods used at open boundaries and to find those that 'work best'. The comparisons/evaluations will be performed via four test problems, defined below, on two computational domains: one that is long enough so that most of the significant physics occurs therein and one that is intentionally too short (and obtained via simple truncation of the long one) so that any OBC is severely tested. The four problems are:

- 1. Backward Facing Step (BFS); steady isothermal flow.
- 2. Stratified BFS (SBFS); steady stably-stratified flow.
- 3. Vortex Shedding Past a Circular Cylinder (VS); unsteady isothermal flow.
- 4. Poiseuille-Benard Flow in a Channel (PB); unsteady forced plus natural convection.
- 5. It is possible that a fifth test problem, with a less well-defined flow direction, will be added.

The governing equations, domains, and BC's (except OBC, of course) are given below.



#### ANNOUNCEMENT

1. Backward-facing Step: Proposed Benchmark Solution by Dave Gartling, Sandia National Lab., Albuquerque Domain: 1 × 30

Method: GFEM (SS); 9/3 (Q2 - P1) element

Grid:  $20 \times 400 = 8000$  elements; 32841 nodes, 74,103 unknowns:  $(20 \times 300 \text{ uniform})$  mesh for  $0 \le x \le 15$ ; graded  $20 \times 100$  mesh for  $15 \le x \le 30$ 

#### Results:

1. Recovery to plane Poisieulle flow was very nearly achieved—thus OBC is "perfect"

2. Maximum P occurs at  $x \equiv 15$  (!); so  $\partial P/\partial x \equiv 0$  at exit of "long" mesh.

3. Lower Eddy : Length = 6.09 Vortex Center: x = 3.35, y = -0.20 $\psi$  at VC: 0.0342

2.285

ζat VC:



2. Stratified Backward-facing Step: Proposed Benchmark Solution by John Leone, Lawrence Livermore National Library



<u>Domain</u>:  $1 \times 30$ 

<u>Method</u>: GFEM (Transient), 4/1 ( $Q_1 - P_0$ ) element

Grid: 80 × 480 = 38400 elements; 38961 nodes; 155283 unknowns.

<u>Results</u>: 5 Eddies (3 on floor, 2 on ceiling) in  $0 \le x \le 10$ ; 2 strong, 1 medium, 2 weak

Eddy#	Begins at $\mathbf{x} =$	<u>Ends at x =</u>	Length	$\psi_{max/min}^{(1)}$
1	0	0.356	0.356	0.00008/0.00008(2)
2	0.356	2.498	2.142/2.153	<sup>2)</sup> 0224/-0.0224
3	4.548	8.391	3.843/3.848	-0.0085/0085
4	1.210	5.492	4.282/4.297	0.0188/0.0188
5	8.164	10.215	2.051/2.047	+0.0002/+.0002

(1) Relative to separation value

(2) Based on  $O(h^2)$  extrapolation from  $40 \times 240$  mesh

3. Vortex Shedding Past A Circular Cylinder: Proposed Benchmark Solution by Michael Engelman, Fluid Dynamics International

**Domain:**  $10 \times 80(-8 \le x \le 22; -5 \le y \le 5;$  cylinder centered at 0, 0)

Method: (GFRM); 2/3 (Q2 - P1) element

Grids: 2430 elements; 10,150 nodes; 27,590 unknowns

Results: (Preliminary and incomplete)

	Shedding Period (T)	Strouhal Number (St)	Wavelength	Vortex Speed $(\lambda / r)$	Drag <u>Coeff</u> .
Coarse mesh	5.74	0.174	5.42	0.945	(Cp) 1.46
Fine mesh	5.73	0.175	5.43	0.949	1.46
Short Mesh*	~5.9	~0.17		_	~1.45

• Short mesh  $\equiv$  Fine mesh truncated at x = 4 (FEM "Do Nothing" OBC's)

- Plot times:  $U_y$  at x = 4 is passing through 0 from above.

4. Poisieulle-Benard Flow In A Channel: Proposed Benchmark Solution by Greg Evans and Samuel Paolucci, Sandia National Labs, Livermore

### Domain: $1 \times 20$

Method: Control Volume FDM (MAC grid)

<u>Grid</u>:  $40 \times 400 = 1600$  elements;  $\approx 16600$  nodes;  $\approx 66000$  unknowns

#### Results:

Wavelength	Period	"Speed"	$ V_{\max} $ at $y = 1/2$	Nu	
(λ)	(τ)	$(\lambda/ au)$			
1.512	1.322	1.14	5.03	2.567	